

FAMIN[®]. Experience, innovation and close cooperation with leading international companies



MT Gear Coupling Introduction

AURE MT Crowned Tooth Gear Couplings Famin was founded in Iran in 2001 as a machining workshop in one of the most industrialized areas with strong steel and iron tradition. Jaure's flexible coupling manufacturing activity as it is known nowadays started back in 2005 with the production Along the last 20 years, the design of Famin gear coupling has been continuously improved Thousands of Famin gear couplings are operating today in the toughest applications such as steel mills, pulp and paper, mining, cement, marine drives, wind turbines, etc. MT gear couplings are the most compact coupling solution for critical applications that demand high torque transmission.



Main advantages of MT gear couplings

Famin® couplings offer maximum torque capacity. This is due to the optimum pitch diameter of the gears, providing reliability.

High permissible hub bore allows more favorable size selection of the coupling for a certain shaft diameter. This offers an important economic saving.

High permissible additional loads for starting and short-circuit peak torque.

Highest gear accuracy and quality thanks to the production improvements obtained with new CNC gear cutting machines and automatic charge systems.

The Famin MT standard range meets the AGMA standard, meaning that the MT coupling sleeves and drilled holes will fit any AGMA coupling halves. This ensures the interchangeability by coupling halves. MT Gear Coupling Introduction

Custom made couplings

Special designs according to customer needs often come from close co-operation with our R&D and Engineering departments. Special solutions are normally based on the use of special seals and alloyed steels subjected to surface hardening treatments, such as:

Nitriding Case carburizing Induction hardening

MT Gear Coupling Description

The MT gear coupling is a steel double-joint coupling. Main function is to transmit torque and at the same time accommodate the misalignment between two shafts.

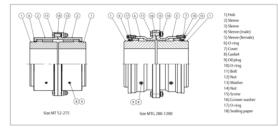
The MT coupling is torsionally stiff and formed by two crowned hubs which engage two flanged sleeves with internal straight parallel teeth (see coupling parts at Fig nº1)

As a result of the teeth curvature, if shafts misalignment occurs, the crowned teeth hubs can oscillate in the flanged sleeve (see Fig $n^{\circ}2$).

It is impossible to have corner pressure even at maximum misalignment. The combined tip and flank centering and fully machined coupling ensure smooth operation.

In case of high rotation speed (circumferential speeds exceeding 40 m/s or sensitive supports to unbalance) dynamic balancing is required.

The teeth are machined with precision gear machines in order to assure uniform contact



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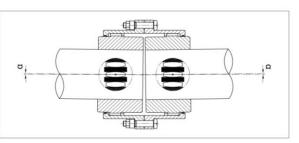


Fig nº1. Coupling components

Fig nº2. Detail of the crowned teeth with angular misalignment



Coupling size for certain drive depends not only on the drive unit power and speed, but also on the misalignment and the type of machines to be coupled.

Three types of misalignment can be effectively accommodated by the MT gear coupling (see Fig. n° 3):

Axial: Shafts are aligned but shaft ends are apart from each other. Parallel Offset: Axes of connected shafts are parallel, but not in the same straight line. Angular: Axes of shafts intersect at center point of coupling, but not in the same straight line

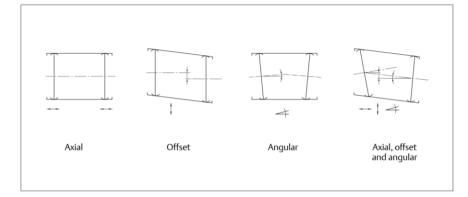


Fig nº3. Shaft misalignment

Proper maintenance of gear couplings is key to avoid early wear of the gear teeth and extend the coupling's life. It is necessary to strictly follow the installation and maintenance instructions.

The MT gear coupling must be filled with grease periodically. Therefore, the lubrication is forced into the teeth by centrifugal force. Seals are provided in the sleeves/covers to prevent any grease leakage. They can be used at -30°C to 70°C temperature range. For other temperature, please contact Famin.

The gear couplings are made in C45/55 material as a standard. If a more compact coupling or higher power ratio is required, Famin Heavy Duty Type (HD) in alloy steel is available in our program.

In addition other special designs with heat treatments such as induction hardening, gas nitriding, case hardening, etc can be delivered on demand. Please contact our engineering or sales departments.

For more detailed information refer to Famin instruction manual



For MT standard design selection, the following data is required:

Coupling Selection

- -PN, Installed or absorbed power (Kw)
- -n, operating speed (rpm)
- -L, d shaft lengths and diameters (mm)
- -DBSE distance between shaft ends (mm)
- -Service requirements (K service factor)
- -Dynamic misalignment (Fa misalignment factor)
- -Additional geometrical or atmospheric restrictions

Torque Capacity varies with speed and dynamic misalignment. A coefficient (f α = Coefficient factor from dynamic misalignment and speed) is required over 0,1 degrees of misalignment and is affected in the following trend:

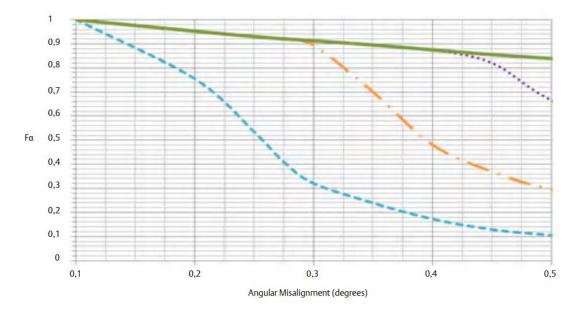
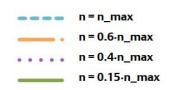


Fig. nº 4 Trend of Coefficient with Dynamic Misalignment and Speed

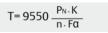


Above graph shows one gear mesh misalignment.

Values shown in the graph are indicative. For accurate calculation or higher angular misalignment please contact Famin



Selection procedure 1) Calculate nominal torque TN (Nm) as follows:



PN= Max. actual power in (Kw)

n= Coupling speed in (rpm.)

K= Service factor Fα = Misalignment factor

Select coupling size from catalogue with nominal torque capacity same or higher than obtained. Check if the peak torque of the application is below the coupling max. torque (TPmax).

2) Check in the catalogue maximum bore capacity for selected coupling. Should shafts be larger than the maximun admissible bore, select next bigger necessary size diameter.3) Check shaft/hub connection is able to transmit the torque. If necessary, extend the hub length.

4) Speed given in catalogue is maximun value for unbalanced couplings. For higher operational speed the coupling must be dynamically balanced and other materials than carbon steels might be used. Please contact Famin engineering for support.

5) Selection Service Factors (K)

The service factor can vary for each application and depending among other on:

a. Type of driving and driven machine

b. Reversing / Non-reversing load

c. Peak torques

Example:

Find a coupling to connect a gearbox with the drum of a conveyor.

Motor power PN = 400Kw.

Peak torque: 7200Nm

Drum speed n = 1.000 r.p.m.

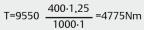
Gearbox shaft d1 = 80mm.

Drum side shaft d2 = 100mm.

Dynamic misalignment < 0,1 degrees. Fα=1

Service Factor for Conveyor Heavy Duty Not Uniformly Fed Assembly K=1.25

Solution:



Nominal torque needs to be checked as a first step. From MT basic design, we would select MT-78. Secondly, we need to check maximum shaft capacity for selected size. This would lead to the selection of MT-112, as the drum shaft diameter is 100 mm.

Check that Peak Torque of application 7200 Nm is below selected coupling limit (28200 in this case).

With this selection, resulting Service factor can be calculated: Since

T application =
$$9950 \frac{P_N}{n} = 9550 \frac{400}{1000} = 3820 \text{Nm}$$

Resulting Service Factor can be calculated as



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Balancing

Coupling Balance requirements and limits are mainly dependent upon the characteristics of the application. For this reason balancing charts should be used as a guide only to assist in determining whether balancing is required or not.

Balancing chart is shown for average applications. For sensitive or critical application, contact Famin for coupling balance requirement.

Balance Chart for non spacer type gear couplings

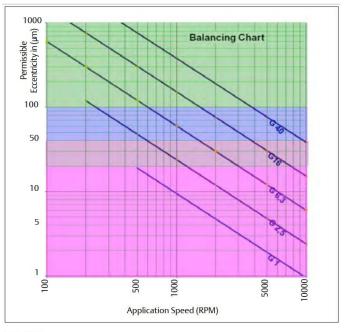
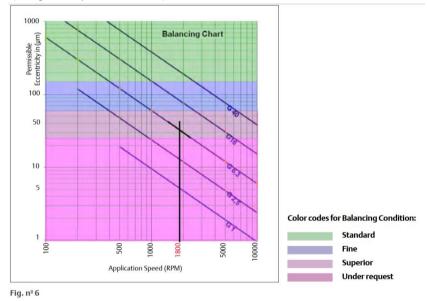


Fig. nº 5

Balance Chart for intermediate shaft couplings up to 1000mm of DBSE (For higher DBSE please refer to Jaure)





Balancing

Minimum applications data required for chart interpretation:

ISO balancing Grade for the coupling. (G)
Application Speed (n)

This information allows to calculate the permissible eccentricity (e permissible) for the coupling

e permissible = 9550.G/n

Where,

e permissible = Permissible eccentricity of center of gravity in μ m.

e coupling = Actual eccentricity of center of gravity of coupling in μ m.

G = Balance Grade in mm/s

n = Application speed in rpm.

In order to satisfy the application requirement, e coupling \leq e permissible.

Balancing Practices

Famin couplings are dynamically balanced in component level or in sub-assembly. In case of sub-assembly balancing level, major components are match-marked to ensure the proper reassembly of the coupling. Hubs are component balanced at finished bore without keyways unless mentioned in the order. On special request from customer, assembly balancing of gear coupling including gear hubs can be performed. Balancing reports will be available for customer under request. **Example**:

Coupling Type – Spacer type Gear Coupling. DBSE – 600 mm Required Balance Quality – 6.3 Application Speed – 1800 rpm

As per the chart, we require to perform the superior balancing for the spacer coupling to achieve the 6.3 balancing grade as per ISO-1940-1.

Alternatively: e permissible = 9550.G/n = 9550. (6.3)/1800 = 33,4 μm

From the chart (Y-axis) for $33,4\,\mu m$ permissible eccentricity, coupling requires Superior balancing.

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Industrial designs

Famin gear couplings are used in a variety of demanding applications since years. Paper mill, cranes, pumps, conveyors and any process in a steel, aluminum or cooper mill are just some of the examples where Famin gear couplings are performing successfully. Numerous designs and coupling sizes are available. The custom-made gear coupling offering is developed in close collaboration with the

customer.

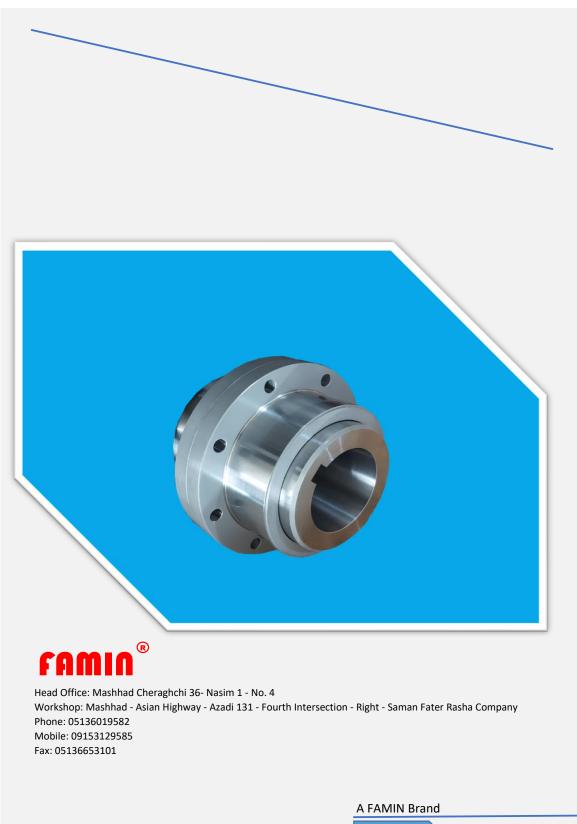


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GEAR COUPLING Size 1-7 $B \longrightarrow B \longrightarrow$														
ONE HUB REVERSED TWO HUBS REVERSED														
Coupling Size	Maximum Bore with Standard	Rating HP / 100	Torque Rating	Peak Torque Rating	Maximum Speed	Dimensions								
	Key	RPM	(lbin.)	(lbin.)	(RPM)	A	В	C	C1	Cw	E	G	0	
1	1 5/8	10	6300	12600	10000	4 9/16	3 3/16	1/8	3/8	5/8	1 11/16	3	2 5/16	
1 1/2	2 3/16	24	15100	30200	7400	6	3 7/8	1/8	9/16	1	2 1/16	3 13/16	3 1/8	
2	2 3/4	50 90	31500 56700	63000	5900 5000	7	4 5/8	1/8	13/16	1 1/2	2 7/16	4 13/16	4	
2 1/2	3 1/4			113400		8 3/8	5 11/16	3/16	29/32	15/8	3 1/32	5 23/32	4 23/32	
3	4	150	94500	189000	4300	9 7/16	6 9/16	3/16	1 1/32	1 7/8	3 19/32	6 23/32	5 5/8	
3 1/2	4 3/4	230	145000	290000	3900	11	7 5/8	1/4	1 5/16	2 3/8	4 3/16	7 3/4	6 5/8	
4	5 3/8	350	221000	442000	3500	12 1/2	8 5/8	1/4	1 7/16	2 5/8	4 3/10	8 31/32	7 1/2	
4 1/2	6	480	300000	600000	3200	13 5/8	9 5/8	5/16	1 5/8	2 15/16	5 3/8	10 3/8	8 1/2	
5	6 3/4	650	410000	820000	2900	15 5/16	10 13/16	5/16	1 11/16	3 1/16	6 1/8	11 3/8	9 1/2	
5 1/2	7 1/2	850	536000	1072000	2700	16 3/4	11 5/8	5/16	1 7/8	3 7/16	6/5/8	12 9/16	10 27/64	
	,					/ ·	-,	-, -,			- , - , -	.,	. ,	
6	8 1/4	1100	693000	1386000	2500	18	13 1/4	5/16	2 5/16	4 5/16	7 3/8	13 7/8	11 3/4	
7	9 1/4	1600	1010000	2020000	2200	20 3/4	14 3/4	3/8	2 3/16	4	8 11/16	15 3/4	13 1/4	
Sizes 5 1/2 , 6	and 7 are only	available wit	th exposed bo	olt sleeves. Ty	pe EB exposed	l bolt sleeves	are standar	d.					SFR Co.	

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